

## Modeling Predictions of the Performance of Tube Bank Heat Exchangers with Phase Change Materials

### PCM Encapsulated Tube Bank Heat Exchanger Model

The analytical model used was from the paper "Analytical model of a PCM-air heat exchanger" by Dubovsky, Ziskind, & Letan. [1]

- Initially based off a thermal resistance network, progression is made from an analytical solution for a single tube to the complete heat exchanger through a series of partial differential equations

### Governing Equation for total heat transfer of the tube bank

$$\frac{Q(\tau)}{Q_0} = \begin{cases} \frac{1}{1 - e^{-b}} - \frac{1}{bN} * \frac{h_f}{h_0} \log\{1 + e^{-b\tau}(e^{b\zeta_f N} - 1)\}, \tau \leq 1 \\ \frac{1}{1 - e^{-b}} - \frac{e^{-b}}{1 - e^{-b}} * \frac{h_f}{h_0} * \frac{\tau - 1}{N} - \frac{1}{bN} * \frac{h_f}{h_0} \log(\theta), 1 \leq \tau \leq \tau_0 \end{cases}$$

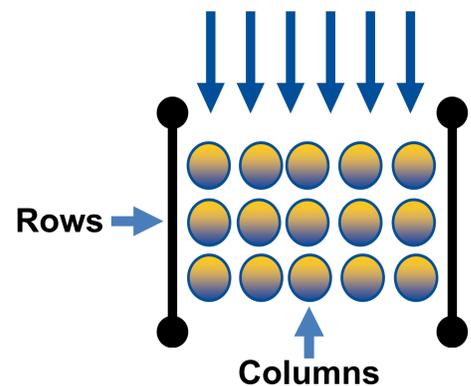
Variable	Variable Representation
$Q_0$	Energy required to be stored in PCM
$\tau$	Dimensionless time
$b$	Constant based on thermal resistance calculations
$N$	Number of tube rows
$h_f$	Heat transfer coefficient from the tubes to the air
$h_0$	Overall heat transfer coefficient
$\zeta_f$	Constant based on the heat transfer coefficients
$\theta$	Constant calculated from $b$ and $\zeta_f$ with $h_f$ & $h_0$

### Analytical Model Design Conditions

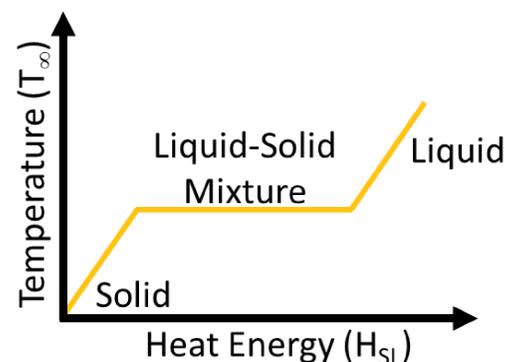
#### Encapsulation Method ( $k_{tube}$ )



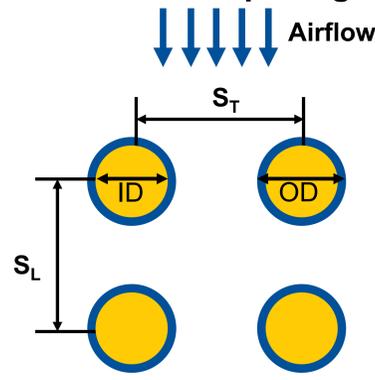
#### Air Temperatures ( $T_{\infty}, \rho_{air}$ )



#### PCM Used



#### Tube Bank Spacing



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### Abstract

As decarbonization of the power grid increases with renewable power generation sources, the intermittent nature of power generation also increases. With intermittent sources of power, there is a greater chance of excess power at specified parts of the day. To ensure that all power generated can be used rather than lost, ways to store excess energy need to be investigated. Currently the most prevalent form of energy storage is through electrochemical batteries. Electrochemical batteries however, are expensive and require added infrastructure for deployment. An alternative to electrochemical batteries is thermal energy storage (TES), that can aid in reducing buildings heating ventilation and air conditioning (HVAC) loads on the power grid. TES is commonly conducted by using phase change materials (PCM) which melt or solidify at specified temperatures. By taking advantage of the latent heat energy stored in the PCM, this energy can be used to condition a space later, reducing the peak thermal loads for a building.

One way to use PCM for TES is through tube bank heat exchangers, which assemble a bundle of PCM encapsulated tubes in a particular spacing. Once the tube bank is configured, air would then pass over the tube bank exchanging heat with the PCM encapsulated tubes. This project focuses on optimizing the tube bank configuration to ensure the time taken to charge & discharge the TES would allow for the thermal battery to be used daily. Predictions for the heat exchangers performance is made through an analytical model while varying specified conditions including the tube bank spacing, encapsulation methods, the specified PCM used, and the incoming air temperatures. By modifying these conditions in the analytical model, estimations can be made before a physical test bed is produced for validation.

### Model Motivation

Modeling was completed to aid in the design of the Ignite project "Investigation of Thermal Energy Storage-Heat Pump Integration for Residential Applications".

- Currently the test bed is in the final steps of construction, and model validation will begin before the end of the semester.



### References

[1] V. Dubovsky, G. Ziskind and R. Letan, "Analytical model of a PCM-air heat exchanger," *Applied Thermal Engineering*, no. 31, pp. 3453-3462, 2011.  
 [2] McMaster-Carr, "Clear Scratch and UV-Resistant Acrylic Round Tube," [Online]. Available: <https://www.mcmaster.com/18532K16/>.  
 [3] ClearTec Packaging, "1/2\" Ultra-Thin Wall Plastic Tube - Length 48\" [Online]. Available: <https://cleartecpackaging.com/plastic-tubing.html>.  
 [4] McMaster-Carr, "Thin-Wall Aluminum Unthreaded Pipe," [Online]. Available: <https://www.mcmaster.com/4561T311/>. [Accessed 4 January 2024].  
 [5] McMaster-Carr, "Copper Tubing for Drinking Water," [Online]. Available: <https://www.mcmaster.com/5175K136-5175K47/>. [Accessed 4 January 2024].  
 [6] Entropy Solutions, "PureTemp 20 Technical Data Sheet," [Online]. Available: <https://puretemp.com/wp-content/uploads/2021/06/PureTemp20TechnicalDataSheet.pdf>.  
 [7] Rubitherm, "RT21 Data Sheet," [Online]. Available: [https://www.rubitherm.eu/media/products/datasheets/Techdata\\_RT21\\_EN\\_31102022.PDF](https://www.rubitherm.eu/media/products/datasheets/Techdata_RT21_EN_31102022.PDF).  
 [8] Rubitherm Technologies, "RT22HC Data Sheet," [Online]. Available: [https://www.rubitherm.eu/media/products/datasheets/Techdata\\_RT22HC\\_EN\\_09102020.PDF](https://www.rubitherm.eu/media/products/datasheets/Techdata_RT22HC_EN_09102020.PDF).  
 [9] Y. A. Cengal and A. J. Ghajar, "Table A-3 Properties of solid metals," in *Heat and Mass Transfer Fundamentals & Applications*, 2015, p. 910.  
 [10] The Engineering Toolbox, "Plastics - Thermal Conductivity Coefficients," [Online]. Available: [https://www.engineeringtoolbox.com/thermal-conductivity-plastics-d\\_1786.html](https://www.engineeringtoolbox.com/thermal-conductivity-plastics-d_1786.html).  
 [11] S. Feja, C. Hanzelmann and R. Kunz, "Measurement of the thermal conductivity of hexadecane," 2018.  
 [12] J. R. Hirsche, N. Kumar, T. Turnaoglu, K. R. Gluesenkamp and S. Graham, "Review of Low-Cost Organic and Inorganic Phase Change Materials with Phase Change Temperature between 0C and 65C," *International High Performance Buildings Conference*, 2021.  
 [13] R. Holmen, M. Lamvik and O. Melhu, "Measurements of the Thermal Conductivities of Solid and Liquid Unbranched Alkanes in the C16 to C19 Range During Phase Transition," *International Journal of Thermophysics*, 2002.

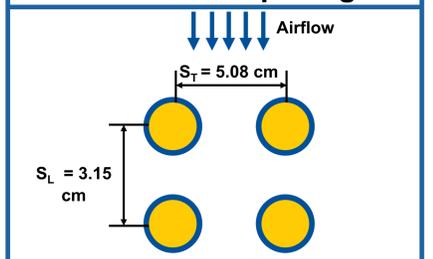
### Physical Test Bed Design Conditions

Encapsulation Material			Incoming Air Temperatures	
Material	Thermal Conductivity [W/mK]	Price per foot	Fan Mode & Flow Rate [L/min]	Average Temperature [°C]
Acrylic	0.20	\$2.79	Medium - 6,711	7.7
PETG	0.20	\$0.45		
Aluminum	238.25	\$23.47	High - 9,090	8.9
Copper	404.68	\$6.55	Super-High - 11,298	9.6

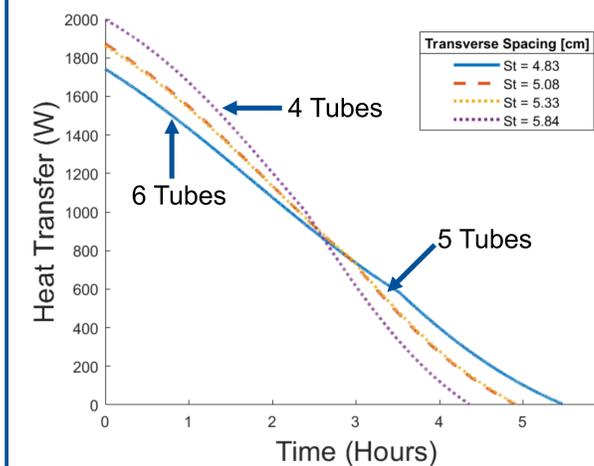
### PCM Selected

PCM Product	Phase Change Temperature [°C]	Heat Storage Capacity [kJ/kg]
PureTemp20	20	171
Rubitherm2 1	21	165
Rubitherm2 2	22	190

### Tube Bank Spacing

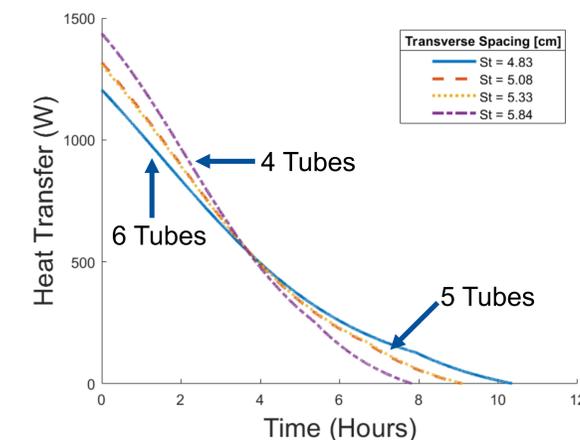


### Analytical Results from Test Bed Conditions - Time to Charge



**Tube Bank Characteristics**  
 Ideal: 5 Tube per Row  
 112 rows

### Design Condition Changes



**Tube Bank Characteristics with Hexadecane**  
 Ideal: 5 Tube per Row  
 84 rows